MECHANICAL ENGINEERING Paper I

Time Allowed: Three Hours

Maximum Marks: 300

20

QUESTION PAPER SPECIFIC INSTRUCTIONS

Please read each of the following instructions carefully before attempting questions

There are **EIGHT** questions divided in **TWO** Sections.

Candidate has to attempt **FIVE** questions in all.

Question Nos. 1 and 5 are **compulsory** and out of the remaining, any **THREE** are to be attempted choosing at least **ONE** from each Section.

The number of marks carried by a question/part is indicated against it.

Wherever any assumptions are made for answering a question, they must be clearly indicated.

Diagrams/Figures, wherever required, shall be drawn in the space provided for answering the question itself.

Unless otherwise mentioned, symbols and notations have their usual standard meanings.

Psychrometric Chart is given on Page No. 33, Steam table is given on Page Nos. 57-61.

Attempts of questions shall be counted in sequential order. Unless struck off, attempt of a question shall be counted even if attempted partly.

Any page or portion of the page left blank in the Question-cum-Answer Booklet must be clearly struck off.

Answers must be written in **ENGLISH** only.

SECTION A

- 1.(a)A wooden block of specific gravity 0.75 floats in water. If the size of the block is 1
mt \times 0.5 mt \times 0.4 mt, find its metacentric height.12
- 1.(b) In a cold winter night with an outside ambient temperature of 2°C, a wall of house steadily loses 30 kJ per minute. If the inner and outer surface temperatures of the wall are maintained at 25°C and 8°C, respectively, determine the rate of energy destruction within the wall in watts.
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- **1.**(c) What are the physical assumptions necessary for a lumped capacity unsteady state heat transfer analysis to be applied? 12
- 1.(d) The brake fuel conversion efficiency of an engine is 30%. The mechanical and combustion efficiencies are 80% and 94%, respectively. The heat losses to the oil and coolant are 60 kW. The chemical energy of the fuel entering the engine is 190 kW. What percentage of this energy becomes (a) brake power; (b) friction power; (c) heat losses; (d) exhaust chemical energy; (e) exhaust sensible energy? 12
- **1.**(e) Briefly explain the working principle of a vortex tube refrigeration system. 12
- 2.(a) The diameter of the horizontal pipe which is 300 mm is suddenly enlarged to 600 mm. The rate of flow of water through this pipe is 0.4 m^3 /sec. If the intensity of pressure in the smaller pipe is 125 KN/m^2 , determine
 - (i) loss of head, due to sudden enlargement.
 - (ii) intensity of pressure in the larger pipe.

(iii) power lost due to enlargement.

2.(b) Argon at 39.85°C with a volume of 0.5 m³ is initially contained in a pistoncylinder (cross sectional area 0.7 m² and height 5.7 m) system with a massless piston loaded with water at 20°C and outside atmosphere (atmospheric pressure, $P_o = 101.203$ kPa) as shown in figure. If the piston just touches the stops, the volume of argon would be 0.8 m^3 . Heat is now added until the temperature of argon reaches 251.85°C. Plot the entire process on *P*-*v* diagram. Assume piston to be adiabatic, determine

- (i) the final pressure inside the cylinder
- (ii) work done (in kJ) and
- (iii) heat transfer during the process (in kJ).

Neglect the volume occupied by the piston and stops.

Take g = 9.807 m/s², specific volume of water = 0.001002 m³/kg. R for argon = 0.2081 kJ/kgK. ω for argon = 0.312 kJ/kgK.



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2.(c) Aluminum fins 1.5 cm wide and 10 mm thick are placed on a 2.5 cm diameter tube to dissipate the heat. The tube surface temperature is 170°C and the ambient fluid temperature is 25°C. Calculate the heat loss per fin for h = 130 w/m²°C. Assume k = 200 w/m°C for aluminum. Use the fin-efficiency curves given below:



3.(a) A four cylinder SI engine has a stroke of 90 mm and a bore of 60 mm, with rated speed of 2800 rpm. The engine is tested at the rated speed against a brake which has a torque arm of 0.356 m. The brake load is 155 N and the fuel consumption is 6.74 l/hr. The specific gravity of the petrol used is 0.735. The net heating value of the petrol used is 44200 kJ/kg. A Morse test is carried out and the cylinders are cut out in the order 1, 2, 3, 4 with corresponding brake load of 111 N, 106.5 N, 104.2 N and 111 N, respectively. Calculate for this speed, the engine torque, the bmep, the brake thermal efficiency, the specific fuel consumption, the mechanical efficiency and the indicated mean effective pressure.

- 3.(b) Describe the working of steam jet-ejector refrigeration system with the help of a neat sketch. Also state the important relations for normal shock in the steam jet refrigeration system.
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- 3.(c) A combination of a heat engine driving a heat pump as shown in figure, takes waste energy at 50°C as a source Q_{w1} to the heat engine rejecting heat at 30°C. The remainder Q_{w2} goes into the heat pump that delivers a Q_H at 150°C. If the total waste energy is 5 MW, find the rate of energy delivered at the higher

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temperature.



- 4.(a) A room with dimensions as 3.5 m wide, 3 m high and 6 m deep, is required to be air conditioned. One of the walls $(3.5 \times 3 \text{ m})$ faces west and contains a double glazed glass window of size 2 m by 1.5 m. There are no heat gains through the rest of the walls. Calculate the sensible, latent and total heat gains. Also, calculate the room sensible heat factor and the required cooling capacity with the following data: Inside conditions: 25° DBT, 50% RH Outside conditions: 45° DBT, 24°C WBT U_{wall} : 1.78 W/m²K $U_{\rm roof}$: 1.316 W/m²K Effective Temperature Difference for wall: 25°C Effective Temperature Difference for roof: 30°C U_{glass} : 3.12 W/m²K Solar Heat gain of glass : 300 W/m²K Internal shading coefficient of glass : 0.86 Occupancy : 4 peisons (90 W sensible heat per person) (40 W latent heat per person) Lighting load : 33 W/m² of floor area Appliance load : 600 W (sensible) + 300 W (latent) Infiltration : 0.5 Air changes per hour State assumptions, if any. 20[Psychrometric chart attached] **4.**(b) Determine the geometric shape factor for a very small disk A_1 and a large parallel disk A_2 located co-axially at a distance L directly above the smaller one. The radius of the large disk may be taken as α . 20An insulated 0.75 kg copper container containing 0.2 kg water, both in **4.**(c) equilibrium at a temperature of 20°C. An experimenter now places 0.05 kg of ice at 0°C in the container. The specific heat of copper is 0.418 kJ/kgK and latent heat of fusion of ice at 0°C is 333 kJ/kg. What will be the temperature (Tf) at the equilibrium condition when all the (i) ice has melted? Compute the entropy generation during the process (J/K). (ii)
 - (iii) What will be the minimum work needed by a stirrer to bring back the temperature of water at 20°C in kJ ?
 - Take specific heat of water as 4.187 kJ/kgK.

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SECTION B

- 5.(a) Explain how the use of a draft tube at the exit of a Francis turbine will increase its efficiency but may initiate the problem of cavitation. 12
- 5.(b) There is a limitation on maximum temperature in gas turbine as its blades have material constraints. While the usual two-wheeler internal combustion engine doesn't have any such constraints; why? Explain. How are such issues with gas turbine blades resolved? Explain with a neat sketch.
- **5.**(c) What is fusible plug? Where it is used and how it works? Explain with a neat sketch. 12
- 5.(d) Explain the working principle of electrostatic precipitator. How can its efficiency be improved? 12
- 5.(e) Describe the working of Solar thermal vapour absorption air conditioning system.
 Also, list the advantages and limitations of LiBr-H₂O and NH₃-H₂O systems. 12
- **6.**(a) A centrifugal pump lifts water from a sump to an overhead reservoir. The static lift is 40 m out of which 3 m is suction lift. The suction and delivery pipes are both of 30 cm diameter. The friction loss in suction pipe is 2.5 m and in delivery pipe it is 7.5 m. The impeller is 0.5 m in diameter and has a width of 3 cm at the outlet. The speed of the pump is 1200 rpm. The exit blade angle is 20°. If the manometric efficiency is 86%, determine the absolute pressures at the suction and delivery ends of the pump. Assume that the inlet and outlet of the pump are at the same elevation. Take atmospheric pressure as 10.10 m of water.
- 6.(b) A steam power plant has high and low pressures of 20 MPa and 10 kPa and one open feed water heater operating at 1 MPa with the exit as saturated liquid. The maximum temperature is 800°C and the turbine has a total power output of 5 MW. Find the fraction of the flow for extraction to the feedwater and the total condenser heat transfer rate. [Steam tables attached]
- **6.**(c) (i) A propeller-type horizontal axis wind turbine has following operating conditions:

Wind speed : 10 m/s Air density : 1.226 Kg/m³, Rotor diameter : 120 m Rotor speed : 50 RPM Coefficient of performance : 40% Calculate:

- (i) Total power density in wind system in w/m^2 .
- (ii) Total wind power in kW.
- (iii) Maximum extractable power in kW.
- (iv) Maximum torque in kN.
- (ii) Explain the various features of wind-diesel hybrid power generation systems.
 Also, describe the types of operational scheduling for diesel unit.
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- 7.(a) A single row impulse steam turbine with a blade speed of 200 m/s and mass flow rate of 4 kg/s develops 300 kW of power. Steam leaves the nozzles at 500 m/s, and the blade velocity coefficient is 0.92. If the steam leaves the turbine blade at such an angle that the absolute velocity at exit is kept minimum, determine nozzle angles, blade angles and diagram efficiency. Draw compound velocity triangles.

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7.(b) An average flow rate of industrial waste water is 1000 m³/day and 4000 mg/litre

of an organic substance with the composition as : $C_{50}H_{75}O_{20}N_5S$. The organic waste is processed in a mesophilic anaerobic digester at 35°C for biogas production with biodegradation efficiency of 95%. Determine the methane production rate. 20

- 7.(c) (i) What are once through boiler? How do they differ from drum boiler? 10
 - (ii) Why are downcomers fewer in number and bigger in diameter, while risers are more in number and smaller in diameter? 10
- 8.(a) How is the degree of reaction of a centrifugal compressor stage defined? Explain analytically how the degree of reaction varies with flow coefficient for different values of impeller exit air angle. Assume zero swirl at the entry. 20
- 8.(b) Consider a gas turbine cycle with two stages of compression and two stages of expansion. The pressure ratio across each compressor stage and each turbine stage is 8 to 1. The pressure at the entrance of the first compressor is 100 kPa, the temperature entering each compressor is 20°C, the temperature entering each turbine is 1100°C. A regenerator is also incorporated into the cycle and it has an efficiency of 70%. Determine the compressor work, the turbine work and the thermal efficiency of the cycle. Take C_{po} as 1.004 kJ/kgK ratio of specific heats as 1.4.
- 8.(c) The border region of Chattisgarh and Maharashtra has many rice mills as this region is suitable for rice crop. Suppose there is a village in this region which has many rice mills, barren land and jungle around. To meet the energy requirements of the rice mills and the village, which types of renewable energy systems would you like to propose? Justify your proposal. 20



GVP-U-MCH

No China		Specific Volume, m ³ /kg				
Press	Temp.	Sat. Liquid	Evap.	Sat. Vapor		
(kPa)	(°C)	v_f	u_{fg}	v _g		
0.6113	0.01	0.001000	206·131	206.132		
1	6.98	0.001000	129·20702	129.20802		
1.5	13.03	0.001001	87.97913	87·98013		
2	17.50	0.001001	67·00285	67.00385		
2.5	21.08	0.001002	54·25285	54·25385		
3	24.08	0.001003	45.66402	45.66502		
4	28.96	0.001004	34·79915	34.80015		
5	32.88	0.001005	28·19150	28.19251		
7.5	40.29	0.001008	19.23674	19.23775		
10	45.81	0.001010	14.67254	14.67355		
15	53.97	0.001014	10.02117	10.02218		
20	60.06	0.001017	7.64835	7.64937		
25	64.97	0.001020	6·20322	6.20424		
30	69·10	0.001022	5·22816	5.22918		
40	75.87	0.001026	3.99243	3.99345		
50	81.33	0.001030	3.23931	3.24034		
75	91.77	0.001037	2.21607	2.21711		
100	99.62	0.001043	1.69296	1.69400		
125	105.99	0.001048	1.37385	1.37490		
150	111.37	0.001053	1.15828	1.15933		
175	116.06	0.001057	1.00257	1.00363		

		Specific Volume, m ³ /kg			
Press.	Temp.	Sat. Liquid	Evap.	Sat. Vapor	
(kPa)	(°C)	v_f	v _{fg}	vg	
850	172.96	0.001118	0.22586	0.22698	
900	175.38	0.001121	0.20306	0.20419	
950	177.69	0.001124	0.20306	0.20419	
1000	179.91	0.001127	0.19332	0.19444	
1100	184.09	0.001133	0.17639	0.17753	
1200	187.99	0.001139	0.16220	0.16333	
1300	191.64	0.001144	0.15011	0.15125	
1400	195.07	0.001149	0.13969	0.14084	
1500	198.32	0.001154	0.13062	0.13177	
1750	205.76	0.001166	0.11232	0.11349	
2000	212.42	0.001177	0.09845	0.09963	
2250	218-45	0.001187	0.08756	0.08875	
2500	223.99	0.001197	0.07878	0.07998	
2750	229.12	0.001207	0.07154	0.07275	
3000	233.90	0.001216	0.06546	0.06668	
3250	238-38	0.001226	0.06029	0.06152	
3500	242.60	0.001235	0.05583	0.05707	
4000	250.40	0.001252	0.04853	0.04978	
5000	263.99	0.001286	0.03815	0.03944	
6000	275.64	0.001319	0.03112	0.03244	
7000	285.88	0.001351	0.02602	0.02737	
8000	295.06	0.001384	0.02213	0.02352	
9000	303.40	0.001418	0.01907	0.02048	
10000	311.06	0.001452	0.01657	0.01803	
11000	318.15	0.001489	0.01450	0.01599	
12000	324.75	0.001527	0.01274	0.01426	
13000	330.93	0.001567	0.01121	0.01278	
14000	336.75	0.001611	0.00987	0.01149	
15000	342-24	0.001658	0.00868	0.01034	
16000	347.43	0.001711	0.00760	0.00931	
17000	352.37	0.001770	0.00659	0.00836	
18000	357.06	0.001840	0.00565	0.00749	
19000	361.54	0.001924	0.00473	0.00666	
20000	365.81	0.002035	0.00380	0.00583	
21000	369.89	0.002206	0.00275	0.00495	
22000	373.80	0.002808	0.00072	0.00353	
22089	374.14	0.003155	0	0.00315	

		Enthalpy, kJ/kg			Entropy, kJ/kg-K		
Press.	Temp.	Sat. Liquid	Evap.	Sat. Vapor	Sat. Liquid	Evap.	Sat. Vapor
(kPa)	(°C)	h_{f}	h_{fg}	h_g	S_f	s _{fg}	Sg
0.6113	0.01	0.00	2501.3	2501.3	0	9.1562	9.1562
1.0	6.98	29.29	2484.89	2514·18	0.1059	8.8697	8.9756
1.5	13.03	54.70	2470.59	2525.30	0.1956	8.6322	8.8278
2.0	17.50	73.47	2460.02	2533-49	0.2607	8.4629	8·7236
2.5	21.08	88·47	2451.56	2540.03	0.3120	8·3311	8.6431
3.0	24.08	101.03	2444.47	2545.50	0.3545	8·2231	8·5775
4.0	28.96	121.44	2432.93	2554.37	0.4226	8.0520	8.4746
5.0	32.88	137·79	2423.66	2561.45	0.4763	7.9187	8.3950
7.5	40.29	168·77	2406.02	2574.79	0.5763	7.6751	8·2514
10	45·81	191·81	2392.82	2584.63	0.6492	7.5010	8.1501
15	53·97	225.91	2373-14	2599.06	0.7548	7.2536	8.0084
20	60.06	251.38	2358·33	2609.70	0.8319	7.0766	7.9085
25	64.97	271.90	2346.29	2618·19	0.8930	6·9383	7.8313
30	69·10	289.21	2336.07	2625·28	0.9439	6.8247	7.7686
40	75.87	317.55	2319.19	2636.74	1.0258	6.6441	7.6700
50	81·33	340.47	2305.40	2645.87	1.0910	6.5029	7.5939
75	91·77	384.36	2278-59	2662.96	1.2129	6.2434	7.4563
100	99.62	417.44	2258.02	2675-46	1.3025	6.0568	7.3593
125	105.99	444.30	2241.05	2685·35	1.3739	5.9104	7.2843
150	111.37	467.08	2226.46	2693.54	1.4335	5.7897	7.2232
175	116.06	486·97	2213.57	2700.53	1.4848	5.6868	7.1717

		Enthalpy, kJ/kg		Entropy, kJ/kg-K			
Press.	Temp.	Sat. Liquid	Evap.	Sat. Vapor	Sat. Liquid Evap. Sat. V		Sat. Vapor
(kPa)	(°C)	h_f	h_{fg}	h _g	S_{f}	s _{fg}	Sg
850	172.96	732·20	2039-43	2771.63	2.0709	4·5711	6.6421
900	175.38	742.82	2031.12	2773.94	2.0946	4.5280	6.6225
950	177.69	753·00	2023.08	2776.08	2.1171	4.4869	6.6040
1000	179.91	762.79	2015-29	2778.08	2.1386	4.4478	6.5864
1100	184.09	781.32	2000-36	2781.68	2·1791	4.3744	6.5535
1200	187.99	798.64	1986-19	2784.82	2.2165	4.3067	6.5233
1300	191.64	814·91	1972.67	2787.58	2.2514	4.2438	6.4953
1400	195.07	830·29	1959.72	2790.00	2.2842	4.1850	6.4692
1500	198·32	844·87	1947·28	2792·15	2.3150	4.1298	6.4448
1750	205.76	878·48	1917·95	2796.43	2.3851	4.0044	6.3895
2000	212.42	908.77	1890.74	2799.51	2.4473	3.8935	6.3408
2250	218.45	936·48	1865-19	2801.67	2.5034	3.7938	6.2971
2500	223.99	962.09	1840.98	2803.07	2.5546	3.7028	6.2574
2750	229.12	985.97	1817.89	2803.86	2.6018	3.6190	6·2208
3000	233.90	1008.41	1795.73	2804·14	2.6456	3.5412	6.1869
3250	238.38	1029.60	1774.37	2803·97	2.6866	3.4685	6.1551
3500	242.60	1049.73	1753.70	2803.43	2.7252	3.4000	6.1252
4000	250.40	1087.29	1714.09	2801.38	2:7963	3.2737	6.0700
5000	263.99	1154·21	1640.12	2794.33	2.9201	3.0532	5.9733
6000	275.64	1213·32	1571.00	2784.33	3.0266	2.8625	5.8891
7000	285·88	1266.97	1505.10	2772·07	3.1210	2.6922	5.8132
8000	295.06	1316.61	1441.33	2757.94	3.2067	2.5365	5.7431
9000	303.40	1363-23	1378.88	2742.11	3.2857	2.3915	5.6771
10000	311.06	1407.53	1317.14	2724.67	3.3595	2.2545	5.6140
11000	318·15	1450.05	1255.55	2705.60	3.4294	2.1233	5.5527
12000	324.75	1491.24	1193.59	2684·83	3.4961	1.9962	5.4923
13000	330.93	1531.46	1130.76	2662·22	3.5604	1.8718	5.4323
14000	336.75	1571.08	1066.47	2637.55	3.6231	1.7485	5.3716
15000	342.24	1610.45	1000.04	2610-49	3.6847	1.6250	5.3097
16000	347.43	1650.00	930·59	2580.59	3.7460	1.4995	5.2454
17000	352.37	1690·25	856.90	2547.15	3.8078	1.3698	5.1776
18000	357.06	1731.97	777·13	2509.09	3.8713	1.2330	5.1044
19000	361.54	1776.43	688·11	2464.54	3.9387	1.0841	5.0227
20000	365-81	1826·18	583.56	2409.74	4.0137	0.9132	4.9269
21000	369-89	1888-30	446.42	2334.72	4.1073	0.6942	4.8015
22000	373.80	2034.92	124.04	2158.97	4.3307	0.1917	4.5224
22080	374.14	2099.26	0	2099.26	4.4297	0	4.4297

Saturated Vapor Water

Temp. (°C)	<i>v</i> (m ³ /kg)	<i>u</i> (kJ/kg)	h (kJ/kg)	s (kJ/kg-K)	<i>v</i> (m ³ /kg)	<i>u</i> (kJ/kg)	<i>h</i> (kJ/kg)	s (kJ/kg-K)
	15000 kPa (342·24°C) 20000 kPa (365·81°C)							;)
Sat	0.01034	2455.43	2610.49	5.3097	0.00583	2293.05	2409·74	4.9269
350	0.01147	2520.36	2692.41	5.4420	_	_		
400	0.01565	2740.70	2975.44	5.8810	0.00994	2619.22	2818·07	5.5539
450	0.01845	2879.47	3156-15	6.1403	0.01270	2806.16	3060.06	5.9016
500	0.02080	2996.52	3308-53	6.3442	0.01477	2942.82	3238-18	6.1400
550	0.02293	3104.71	3448.61	6.5198	0.01656	3062.34	3393-45	6.3347
600	0.02491	3208.64	3582.30	6.6775	0.01818	3174.00	3537.57	6.5048
650	0.02680	3310.37	3712.32	6.8223	0.01969	3281.46	3675.32	6.6582
700	0.02861	3410.94	3840.12	6·9572	0.02113	3386.46	3809.09	6.7993
800	0.03210	3610.99	4092.43	7.2040	0.02385	3592.73	4069.80	7.0544
900	0.03546	3811.89	4343.75	7.4279	0.02645	3797.44	4326.37	7.2830
1000	0.03875	4015·41	4596.63	7.6347	0.02897	4003.12	4582·45	7.4925
1100	0.04200	4222·55	4852.56	7.8282	0.03145	4211.30	4840.24	7.6874
1200	0.04523	4433·78	5112·27	8·0108	0.03391	4422·81	5100.96	7.8706
1300	0.04845	4649.12	5375·94	8.1839	0.03636	4637·95	5365·10	8.0441

MECHANICAL ENGINEERING Paper II

Time Allowed: Three Hours

Maximum Marks: 300

Question Paper Specific Instructions

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SECTION A

Q1. (a) Find the centroidal coordinates of an area bounded by the following curves:

- y = 4 x = 20 $y^2 = 4x$ 12
- (b) Draw the shear force and bending moment diagrams for the following beam:



(c) Define kinematic chain.Find all the inversions of the chain shown in the figure.



(d) The rotor of a turbojet engine has a mass of 250 kg and a radius of gyration of 25 cm. The engine rotates at a speed of 12000 rpm in the clockwise direction if viewed from the front of the aeroplane. The plane while flying at 1500 km/hr turns with a radius of 2 km to the right. Compute the gyroscopic moment the rotor exerts on the plane structure. Also, determine whether the nose of the plane tends to rise or fall when the plane turns.

12

12



- (e) A bolt is loaded with an axial pull of 12 kN along with a transverse shear force of 6 kN. Find the diameter of the bolt required, according to
 - (i) the maximum principal stress theory.
 - (ii) the maximum shear stress theory.

Take permissible tensile stress at elastic point = 100 MPa, Poissons ratio = 0.3. 12

Q2. (a) A rectangular case is loaded with uniform vertical thin rods such that when it is full, as shown in the figure (a), the case has a total weight of 1000 N. The case weighs 100 N when empty and has a coefficient of static friction 0.3 with the floor as shown in the diagram. A force T of 200 N is maintained on the case. If the rods are unloaded from the left end as shown in the figure (b), what is the limiting value of x for equilibrium to be maintained? 20



(b) What is planetary or epicyclic gear train ? In the gear train shown in the figure, the wheel gear C is fixed. The gear B is connected to the input shaft, and gear F is connected to the output shaft. The arm A, carrying the compound gears D and E, turns freely on the output shaft. If the input shaft speed is 1000 rpm in the counter clockwise (ccw) direction when seen from the right, determine the speed of the output shaft. The number of teeth on each gear is indicated in the figure. 20



(c) An internal expanding shoe brake has a diameter of 320 mm and a width of 30 mm. The actuating forces are equal. The maximum pressure is not to exceed 80 kN/m². In reference to the figure given below, the values of various parameters are as follows:

 $\phi_1 = 15^{\circ}, \phi_2 = 145^{\circ}, a = 220 \text{ mm}, c = 125 \text{ mm}.$

Determine the actuating force and the braking torque if the value of the coefficient of friction is 0.32. 20



- **Q3.** (a) Three wires each of 5 mm diameter are used to lift a load of W = 7500 N. An indicative diagram is shown below. The unstressed lengths of the three wires are 18 m, 17.997 m and 17.994 m. Find
 - (i) the stress in the longest wire.

(ii) the stress in the shortest wire if the load is reduced to 2000 N. 20 Take $E = 2.1 \times 10^5$ N/mm².



- (b) Free vibration amplitude of a 500 kg machine mounted on an isolator consisting of a spring and damping element (viscous) is shown in the figure. The time period is also shown in the figure. Find out the characteristics of the isolator i.e.,
 - (i) natural frequency of the system.
 - (ii) stiffness of the spring.
 - (iii) damping coefficient of the isolator.



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(c) A shaft is supported on two bearings 0.8 m apart as shown in the figure. A pulley of diameter 0.2 m is mounted on the shaft which takes the power from vertical belt drive. The shaft also carries another pulley, which transmits power to a machine. The diameter of the pulley is 0.4 m and it is placed to the right hand side of the right bearing at a distance of 0.3 m from the bearing. Both the pulleys contain flat belt at right angles to each other having tension ratio of 2.5 : 1. If the maximum tension in the belt is limited to 3000 N, find the shaft diameter.

Given : Allowable shear stress = 44 MN/m^2 .

 k_{b} = Combined shock and fatigue factor applied to bending = 1.4

 $k_{\rm f}$ = Combined shock and fatigue factor applied to torsional moment = 1.2



- **Q4.** (a) A simply supported beam of length 6 m is loaded with a couple-moment of 20 kN-m at the right end as shown in the figure. Determine
 - (i) the equation for elastic deflection.
 - (ii) the maximum deflection,
 - (iii) the slope at the ends.

Take $E = 2.1 \times 10^5$ N/mm², I = 3000 cm⁴.



(b) The turning moment diagram for a multicylinder engine is to be drawn to a vertical scale of 1 mm = 650 Nm and a horizontal scale of 1 mm = 4.5°. The areas above and below the mean torque line are -28, +380, -260, +310, -300, +242, -380, +265 and -229 mm².

The fluctuation of speed is limited to \pm 1.8% of the mean speed which is 400 rpm. The density of the rim material is 7000 kg/m³ and the width of the rim is 4.5 times its thickness. The centrifugal stress in the rim material is limited to 6 N/mm². Neglecting the effect of boss and arm, determine the diameter and cross-section of the flywheel rim. The turning moment diagram may be drawn free-hand.

- (c) Two flat plates subjected to a tensile force P are connected together by means of double-strap butt joint as shown in the figure. The force P is 250 kN and the width of the plate w is 200 mm. The rivets and plates are made of same steel and the permissible stresses in tension, compression and shear are 70, 100 and 60 N/mm², respectively. Calculate
 - (i) the diameter of the rivets,
 - (ii) the thickness of the plates,
 - (iii) the dimension of the seam viz. p, p_t and m,
 - (iv) the efficiency of the joint.



SECTION B

- Q5. (a) What do you mean by Atomic Packing Factor? Find the volume of unit cell and atomic packing factor for face-centred cubic structure having atomic radius 'R'.
 - (b) A pipe of annealed steel with internal diameter of 60 mm and wall thickness of 3.0 mm is to be reduced down to the internal diameter of 54 mm and wall thickness 2.0 mm. Die angle used for pipe drawing is 30°, $\mu = 0.1$ and draft = 3.1. Compare the drawing force on the plug and the movable mandrels. Given: $\sigma_0 = 240 \text{ N/mm}^2$. 12
 - (c) (i) Write the basic assumptions of Linear Programming Problem.
 - (ii) There are two suppliers X and Y producing a product used in automobiles. The suppliers X and Y can produce maximum 100 and 60 units of the product per day, respectively. Buyers A and B consume 50 and 110 units of the product per day, respectively. The transportation costs from supplier X to buyers A and B are Rs. 30 and Rs. 50 per unit of the product, respectively. Similarly, the transportation costs from supplier Y to buyers A and B are Rs. 20 and Rs. 80 per unit of the product, respectively. Formulate the Linear Programming Problem to minimize the transportation cost. 4+8
 - (d) List down the requirements that govern the selection of actuator sizing for a motion axis in a mechatronic equipment. Will the requirements remain same for different types of actuators? Justify the answer.
 - (e) Discuss the importance of Fault Tree Analysis (FTA). Construct a FTA for a windowless room that contains one switch and two light-bulbs. Assume that the top fault event is dark room. 12
- Q6. (a) (i) Discuss the applications of straight polarity and reverse polarity in welding.
 - (ii) Write the functions of flux used in welding process.
 - (iii) Calculate the melting efficiency in the case of arc welding of steel with a potential 25 V and current 180 Amp. The travel speed is 4 mm/sec and cross-sectional area of the joint is 16 mm². Heat required to melt steel may be taken as 12 Joule/mm³ and heat transfer efficiency as 0.80.

5+5+10

(b) (i) Differentiate between True stress-strain and Engineering stress-strain,

using a stress-strain diagram for mild steel.

- (ii) A cylindrical specimen of alloy steel having an original diameter of 12 mm is subjected to a tensile load of 60 kN. If the instantaneous cross-sectional diameter of 10 mm and elongation observed is 10%, determine the true stress and strain hardening exponent 'n' in true stress-strain diagram. The value of tensile strength (K) is given as 1035 MPa. 8+12
- (c) A rotational mechanical system representing motor trailer is shown below. Derive the system of differential equation and the transfer function. 20 where, *T* is a torque applied to the free end
 - $heta_1$ is an angular displacement of rotor with moment of inertia J_1 ,
 - B_1 is the damping coefficient of torsional damper
 - K_1 is the torsional spring stiffness
 - K_2 is the torsional spring stiffness
 - B_2 is the torsional damping coefficient
 - θ_2 is the angular displacement of rotor with moment of inertia J_2 .



- Q7. (a) (i) The demand for an item in a company is observed as 15,000 units per year and the production capacity of the plant is 2,000 units per month. The set-up cost is Rs. 800 and the inventory holding cost is Rs. 25 per unit per year. The shortage cost of one unit is given as Rs. 250 per year. Determine the economic batch quantity and the number of shortages. Also, determine the cycle time, production time, and utilization percentage.
 - (ii) Discuss the classification of inventory items such as ABC analysis, SDE, VED and FSN.
 10+10
 - (b) For the below given 3 DOF (Degree of Freedom) arm, determine the forward kinematic model, using D-H algorithm. Clearly indicate the assumptions, if any, to derive the kinematic model. 20



(c) As an engineer, you were supplied with the two tapered parts as shown below whose taper is to be measured using standard balls and rollers.



10

Elaborate the steps necessary to measure the taper angles of these parts. Provide the list of instruments and materials required, with detailed mathematical derivation for the measurement. Specify the precautions you need to take while measuring the angles. 20

- **Q8.** (a) (i) The position of a point P of a rigid body B is located at $B_p = [1, 2, 3]^T$. Determine the global position after rotation 30° about X-axis and then 45° about Y-axis.
 - (ii) If a homogeneous transformation matrix T is given by

$\int c\theta$	$-c\alpha s\theta$	$s \alpha s \theta$	$ac\theta$
$s\theta$	$c \alpha c \theta$	$-s\alpha c\theta$	$as\theta$
0	sα	cα	d
0	0	0	1

Determine its inverse, using concepts of rigid body motion.

- (b) (i) Discuss the failure rate model having Burn-in period, Useful life period and Wear-out period.
 - (ii) Show the relationship between failure rate (λ), reliability (R), pdf(f) and cdf(F).
 - (iii) Twenty machines have been operated for 100 hours. One machine fails in 70 hours and another in 80 hours. What is the mean time between failure and reliability at 500 hours? Assume the constant failure rate for the above machines.
- (c) (i) The armature controlled DC motor has the following ratings:

 $K_{\rm p} = 0.08$ Nm/A, Maximum current = 3 A

 $K_{\rm e} = 0.06$ V rad/s, Maximum speed = 600 rad/s

Armature resistance = 1.5Ω

Determine the maximum output torque, maximum output power, maximum armature voltage and no-load motor speed. Assume that the frictional torque is zero. 10

 (ii) What are the important components of Programmable Logic Controllers (PLCs)? List down the differences between PLCs and computers in terms of program and memory used, power supply and its disruption, and flexibility in operation.